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TECHNICAL REPORT ARLCB-TR-83041

PREDICTION OF RESIDUAL STRESSES IN AN AUTOFRETTAGED THICK-WALLED CYLINDER

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NOVEMBER 1983



US ARMY ARMAMENT RESEARCH AND DEVELOPMENT CENTER
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4. TITLE (end Subtitle) PREDICTION OF RESIDUAL STRESSES IN AN AUTOFRETTAGED THICK-WALLED CYLINDER	S. TYPE OF REPORT & PERIOD COVERED Final 6. PERFORMING ORG. REPORT NUMBER
7. AUTHOR(s) Peter C. T. Chen	8. CONTRACT OR GRANT NUMBER(s)
9. PERFORMING ORGANIZATION NAME AND ADDRESS US Army Armament Research & Development Center Benet Weapons Laboratory, DRSMC-LCB-TL Watervliet, NY 12189	10. PROGRAM ELEMENT, PROJECT, TASK AREA & WORK UNIT NUMBERS AMCMS NO. 6111.02.H600.011 PRON NO. 1A325B541A1A
11. CONTROLLING OFFICE NAME AND ADDRESS	12. REPORT DATE
US Army Armament Research & Development Center Large Caliber Weapon Systems Laboratory Dover, NJ 07801	November 1983 13. NUMBER OF PAGES 8
14. MONITORING AGENCY NAME & ADDRESS(if different from Controlling	Office) 15. SECURITY CLASS. (of this report) Unclassified 15a. DECLASSIFICATION/DOWNGRADING SCHEDULE

16. DISTRIBUTION STATEMENT (of this Report)

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17. DISTRIBUTION STATEMENT (of the abstract entered in Block 20, if different from Report)

18. SUPPLEMENTARY NOTES

Presented at the IX AIRAPT Conference on High Pressure in Science and Technology, State University of New York at Albany, 24-29 July 1983.

19. KEY WORDS (Continue on reverse side if necessary and identify by block number)

Autofrettaged Tube Residual Stress Bauschinger Effect Strain-Hardening Reverse Yielding

20. ABSTRACT (Continue on reverse side if necessary and identify by block number)

Most of the earlier results for residual stresses are based on the assumption of elastic unloading. In this report, the prediction of residual stresses for the case of reverse yielding including the combined Bauschinger and hardening effect will be reported for an autofrettaged thick-walled cylinder. The Bauschinger effect factor is varying as a function of overstrain. The strain-hardening effect is considered with different parameters used for loading and unloading process. The new results indicate that the influence of the combined Bauschinger and hardening effect on residual stress distribution is significant.

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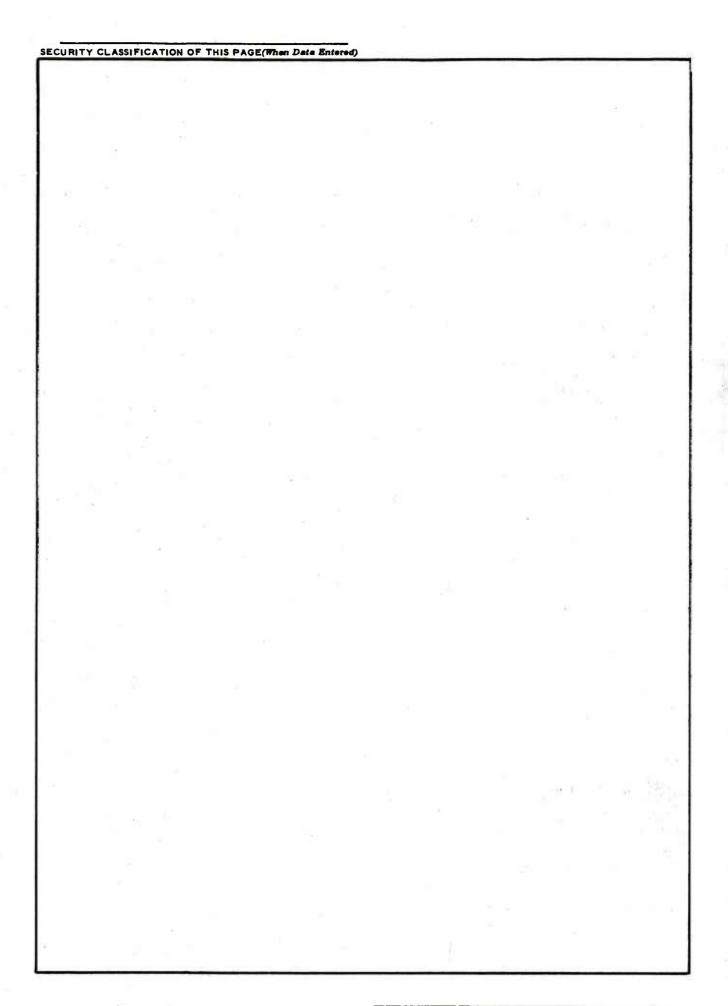


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INTRODUCTION

To increase the maximum pressure a cylinder can contain, it is common practice to produce a more advantageous stress distribution involving residual compressive hoop stresses near the bore by autofrettage treatment of the cylinder prior to use (ref 1). The determination of residual stresses is important in the stress intensity factor calculation (ref 2) and the fatigue life estimation (ref 3). There is, however, considerable disagreement among solutions obtained by different investigators for the residual stress distribution in the cylinder after the autofrettage process (refs 4-6). discrepancy in residual stress is a result of different mathematical methods, end conditions, and material models. Different assumptions for the material properties such as compressibility, yield criterion, flow rule, hardening rule, Bauschinger effect, etc. can lead to many material models. Most of the earlier solutions for residual stresses were based on the assumption of elastic unloading and only a few considered reverse yielding (refs 5-7). unloading with reversed yielding, there is no general agreement in the literature over which material model should be used. Many plasticity theories have been proposed and reviewed (ref 8), yet no theory is completely adequate. In particular, it seems that no theoretical model has been given to represent accurately the actual material behavior in a high strength steel (ref 9) as reported by Milligan, Koo, and Davidson.

In this report a new theoretical model is used with one attempt to give a close representation of the actual loading/unloading behavior in a high

References are listed at the end of this report.

strength steel. The Bauschinger effect factor is treated as a function of overstrain. The strain-hardening effect is taken into account with different parameters used for loading and unloading process. The application of this model to the prediction of residual stresses in an autofrettaged thick-walled cylinder is reported.

THEORETICAL MODEL

Figure 1 shows the stress-strain curve during loading and unloading after overstrain in tension. The stress-strain curve during loading can be replaced with sufficient accuracy by a bilinear elastic-plastic model as shown also in Figure 1. For the plastic portion, the yield stress (σ) is related to the plastic strain (ϵ^{ρ}) by

$$\sigma/\sigma_{O} = 1 + m\zeta/(1-m)$$
 and $\zeta = (E/\sigma_{O})\epsilon^{O}$ (1)

where the Young's modulus (E), tangent modulus (mE), initial yield stress (σ_0) , and the Poisson's ratio (ν) are the material constants.

Choosing a new coordinate system (σ', ϵ') with the origin at the point before unloading, we have for the plastic portion of the reverse yielding curve

$$\sigma'/\sigma_O = \sigma_O'/\sigma_O + m'\zeta'/(1-m')$$
 and $\zeta' = (E/\sigma_O)\epsilon'^O$ (2)

where m'E is the slope of the reverse yielding curve, $\epsilon^{,\rho}$ is the additional plastic strain during unloading and σ_0 , is the linear drop in stress until reverse yielding begins. According to Eq. (1) and experimental data [9], σ_0 can be expressed as a function of plastic strain just prior to unloading by

$$\sigma_{O}'/\sigma_{O} = [1 + m\zeta/(1-m)] [1 + f(\zeta)] = g(\zeta)$$
 (3)

where $f(\zeta)$ is a function of pre-strain as shown in Figure 2.

ANALYTICAL SOLUTION

Consider a thick-walled cylinder, internal radius a and external radius b, which is subjected to internal pressure p. The material is assumed to be elastic-plastic, obeying the Tresca's yield criterion, the associated flow theory, and a bilinear hardening rule for loading and unloading. The elastic-plastic interfaces before and after unloading are represented by ρ and ρ ', respectively. The elastic-plastic solution during loading and unloading can be obtained. Due to space limitations, only some of the final results are given below. During elastic-plastic loading, the equivalent plastic strain and the tangential stress in a \leq r \leq ρ can be calculated by

$$\zeta = \beta_1(\rho^2/r^2-1) \tag{4}$$

$$\sigma_{\theta}/\sigma_{0} = \frac{1}{2} (1 + \rho^{2}/b^{2}) + \frac{1}{2} \beta_{2}(\rho^{2}/r^{2}-1) - (1-\beta_{2})\log \rho/r$$
 (5)

where

$$\beta_1 = (1-m)/[m + \frac{\sqrt{3}}{2} \frac{(1-m)}{(1-\nu^2)}], \quad \beta_2 = m\beta_1/(1-m)$$
 (6)

During elastic-plastic unloading, the additional equivalent plastic strain and the tangential stress in a \leq r \leq ρ' can be calculated by

$$\zeta' = \beta_1'[(\rho'/r)^2g(\zeta_0') - g(\zeta_r)]$$
 (7)

$$\sigma_{\theta}'/\sigma_{o} = p/\sigma_{o} - \sigma'/\sigma_{o} + \frac{1}{2} (\beta_{2}'/\beta_{1})(\rho'/\rho)^{2} (\zeta_{r} - \zeta_{a})g(\zeta_{\rho}')$$

$$-\frac{1}{2}\left(1-\beta_{2}'\right)\int_{\zeta_{\mathbf{r}}}^{\zeta_{\mathbf{a}}}g(\zeta)(\zeta+\beta_{1})^{-1}d\zeta \tag{8}$$

where

$$\beta_1' = (1-m')/[m' + \frac{\sqrt{3}}{2} \frac{(1-m')}{(1-\nu^2)}], \quad \beta_2' = m'\beta_1'/(1-m')$$
 (9)

RESULTS AND DISCUSSION

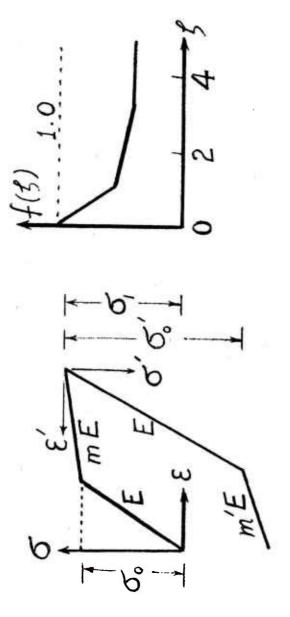
The residual stress system, which will be denoted by two primes, is the sum of the system produced by loading and that produced by unloading, i.e.,

$$\sigma_{\theta}^{"} = \sigma_{\theta} + \sigma_{\theta}^{"} \tag{10}$$

The numerical results for an open-end thick-walled cylinder with b/a=2, $\nu=0.3$, m=0.01 are shown in Figures 3 and 4, for $\rho/a=1.6$ and 2.0, respectively. The other material parameters used in three cases are (a) m'=0.01, f=1; (b) m'=0.01, $f=0.55-0.06\zeta$; (c) m'=0.3, $f=0.55-0.06\zeta$. The first case represents isotropic hardening model with no Bauschinger effect (ref 5). According to this model, there is no reverse yielding. The second case shows the Bauschinger effect but almost no hardening during loading and unloading. The result is very close to that in Reference 7. The third case is a close representation of the actual loading/unloading behavior in a high strength steel (ref 9). The result is shown in solid curve in Figures 3 and 4. The new results indicate that the influence of the combined Bauschinger and hardening effect on the residual stress distribution is significant.

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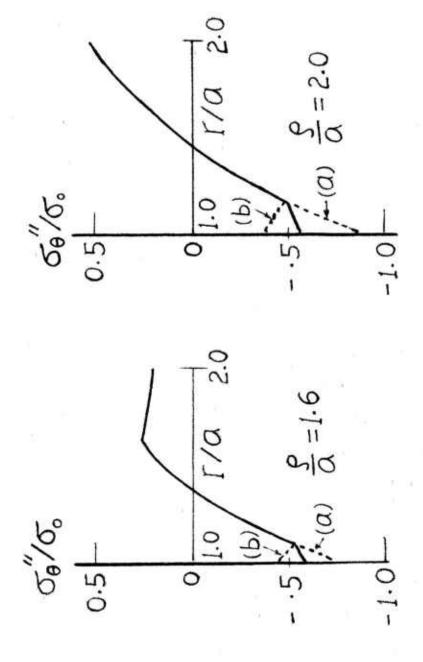


Figure 4. Residual Stress in a Thick-Walled Cylinder (100 percent overstrain).

Residual Stress in a Thick-Walled Cylinder (60 percent

Figure 3.

overstrain).

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